

# Towards Material Modelling in Physical Models Using Digital Waveguides

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## Abstract

Digital Waveguides have been used extensively for musical instrument and room acoustics modelling. They can be used to form simplistic models for ideal wave propagation in one, two and three dimensions. Models in 1-d for string

we find that waves travel with a speed which varies with frequency according to  $c(w) = \sqrt{w/a}$ , where  $a = \frac{\rho A}{EI}$ . Thus wave speed increases (from zero) with frequency. By adding a bar like term to (3) we may represent a stiff string as follows.

$$\frac{\partial^2 y}{\partial t^2} = \frac{F}{\rho} \frac{\partial^2 y}{\partial x^2} - \frac{EI}{\rho A} \frac{\partial^4 y}{\partial x^4}, \quad (5)$$

where now the restoring forces are due to tension and bending stiffness. Again by considering harmonic solutions to (5) we may derive an expression for the frequency dependent wave speed in a stiff string.

$$c(w)^2 = \frac{F}{2\rho} + \sqrt{\left(\frac{F}{2\rho}\right)^2 + \frac{\rho A}{EI} w^2}. \quad (6)$$

Note that when the stiffness is removed that (6) reduces to the case of the ideal string, and that when all tension is removed we reduce to the case of the ideal beam. We also note that for low frequencies the speed approximates that of the string, but that as frequency increases the speed of wave travel becomes more bar like. Finally, we observe that the introduction of a constant tension to a beam results in a non-zero wave speed at zero frequency.

## 2.2 A string on a Viscoelastic Foundation

In this short section we show how it is possible to add new terms to the PDE's described previously in order to introduce dispersion and frequency dependent loss. Firstly we consider placing a string on a purely elastic foundation, which may be thought of as laying the string on bed of springs (Graff, 1975). The governing equation is now

$$F \frac{\partial^2 y}{\partial x^2} - Gy = \rho \frac{\partial^2 y}{\partial t^2}, \quad (7)$$

where the new parameter is the foundation stiffness  $G$ . By again trying harmonic solutions we find the following relationship between frequency and wave number,

$$w^2 = c_0^2 \left( k^2 + \frac{G}{F} \right), \quad (8)$$

where  $c_0 = \sqrt{F/\rho}$  is the wave speed in the absence of foundation stiffness. By considering frequency against wave number we are able to predict both the fundamental frequency, and then the relative positions of each subsequent harmonic. A graph of this relationship is shown in Figure 1 and shows that the fundamental increases with frequency and that as the wavenumber increases, the resonant peaks will approach a harmonic series equivalent to the string in the absence of foundation stiffness.

We now propose a damped string obtained by including a resistive force to the motion resulting in the following governing equation,

$$F \frac{\partial^2 y}{\partial x^2} - g \frac{\partial y}{\partial t} = \rho \frac{\partial^2 y}{\partial t^2}, \quad (9)$$

where  $g$  is the resistive coefficient (Graff, 1975). This process can be thought of in the same terms as for the elastic

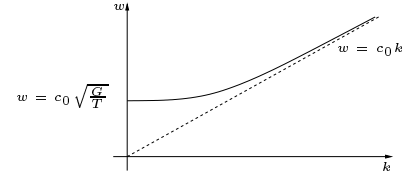


Figure 1: Frequency against wavenumber for a string on an elastic sub-base.

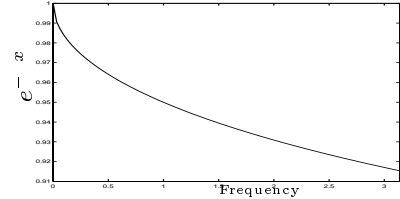


Figure 2: Freq-dependent Damping for String on Viscous Sub-Grade.

foundation, only with dash-pots replacing springs. This time however the damping prohibits the free propagation of harmonic waves, however we may consider solutions of the form  $y = Ae^{-\alpha x} e^{j(kx - wt)} = Ae^{j[(k+j\alpha)x - wt]}$ . Thus we have dispersive travelling waves which also include frequency dependent damping. Solving (9) for these solutions yields

$$k = M^{1/2} \cos(\phi/2), \quad \alpha = M^{1/2} \sin(\phi/2), \quad (10)$$

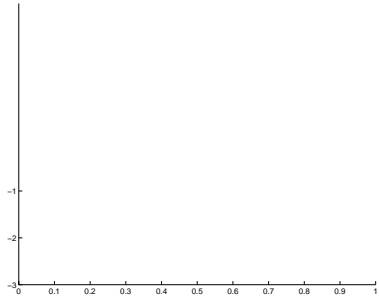
for

$$M = \frac{w}{F} (g^2 + \rho^2 w^2)^{1/2}, \quad \phi = \tan^{-1} \left( \frac{g}{\rho w} \right).$$

We see that  $w$  will dominate for large values of the frequency, so that in any practical situation, the dispersion is minimal at low frequencies, and negligible elsewhere. We should also note that the damping term will cause damping of a lowpass nature. Shown in Figure 2 is the damping term for some typical simulation values.

## 3 Incorporating Bending Stiffness into Waveguide Models





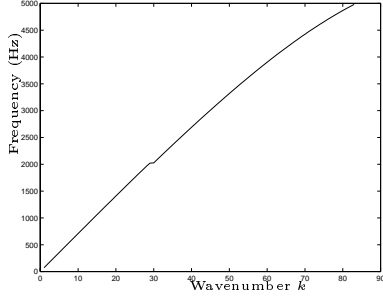


Figure 8: Frequency versus wavenumber for DWN bar model.

attached with impedance  $R_s$ . Be innin with the junction velocity equation for junction  $j$ , we have

$$\begin{aligned}
V_j(n) &= \frac{2}{R} [V_{j,1}^+(n) + V_{j,2}^+(n) + R_s V_{j,s}^+(n)] \\
&= \frac{2}{R} [V_{j-1,2}^-(n-1) + V_{j+1,1}^-(n-1) - R_s V_{j,s}^-(n-1)] \\
&= \frac{2}{R} [V_{j-1}(n-1) + V_{j+1}(n-1) - R_s V_j(n-1)] \\
&\quad - \frac{2}{R} [V_{j-1,2}^+(n-1) + V_{j+1,1}^+(n-1) - R_s V_{j,s}^+(n-1)] \\
&= \frac{2}{R} [V_{j-1}(n-1) + V_{j+1}(n-1) - R_s V_j(n-1)] \\
&\quad - \frac{2}{R} [V_{j,1}^-(n-2) + V_{j,2}^-(n-2) + R_s V_{j,s}^-(n-2)] \\
&= \frac{2}{R} [V_{j-1}(n-1) + V_{j+1}(n-1) - R_s V_j(n-1)] \\
&\quad - V_j(n-2),
\end{aligned}$$

which is equivalent to

$$\begin{aligned}
&V_j(n+1) - 2V_j(n) + V_j(n-1) = \\
&\frac{2}{R} [V_{j-1}(n) - 2V_j(n) + V_{j+1}(n)] - 4\frac{R_s}{R} V_j(n),
\end{aligned}$$

where  $R = 2 + R_s$  is the total junction impedance. Now, recallin equation (7) we may write a FD for the system as

$$\begin{aligned}
&V_j(n+1) - 2V_j(n) + V_j(n-1) = \\
&\mu \frac{F}{\rho} [V_{j-1}(n) - 2V_j(n) + V_{j+1}(n)] - T^2 \frac{G}{\rho} V_j(n),
\end{aligned}$$

where  $T$  is the time step,  $\Delta$  is the spatial step, with  $\mu = \frac{T^2}{\Delta^2}$ . Thus we must fix

$$\begin{aligned}
\frac{2}{R} &= \frac{\mu F}{\rho}, \\
\frac{4R_s}{R} &= \frac{GT^2}{\rho}.
\end{aligned} \tag{17}$$

By fixin the time step  $T$ , then solC1I9T19msolC1W1H1BPC1IMtrueIDEIBTvH1BPC1reT191Tf30 .1-9901.1Tf30 .1-9901.1(s)TjETq1

1 and this time setting the third impedance to  $R_d$  we may follow a similar formulation as before. This time we find that

$$\begin{aligned} V_j(n+1) - 2V_j(n) + V_j(n-1) = \\ \frac{2}{R} [V_{j+1}(n) - 2V_j(n) + V_{j-1}(n)] \\ - \frac{2R_d}{R} [V_j(n) - V_j(n-1)] \end{aligned}$$